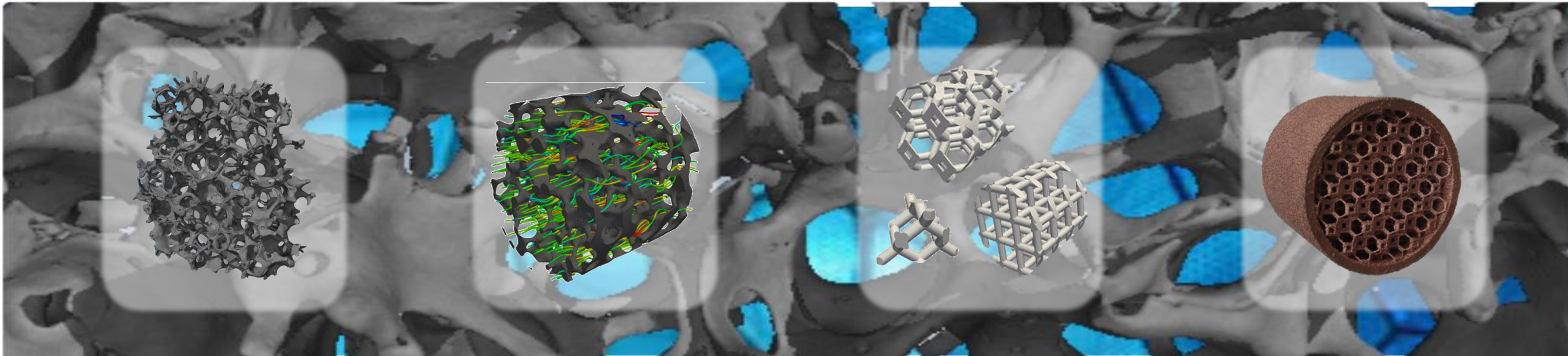


Hydrodynamics and heat transport in highly porous open-celled structures

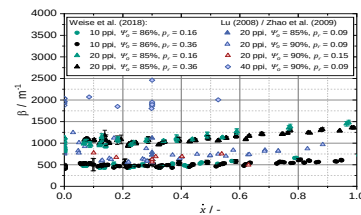
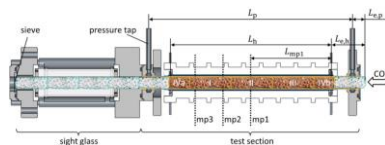
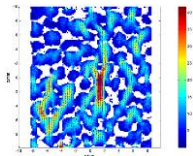
Dr.-Ing. Benjamin Dietrich



Topics on cellular materials @ TVT

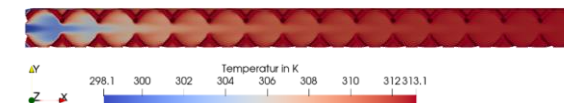
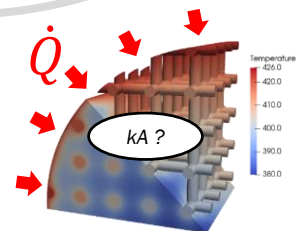
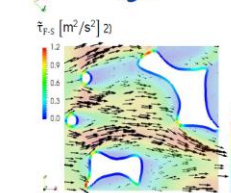
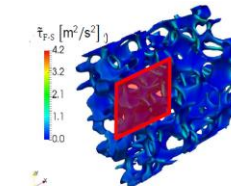
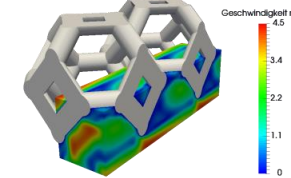
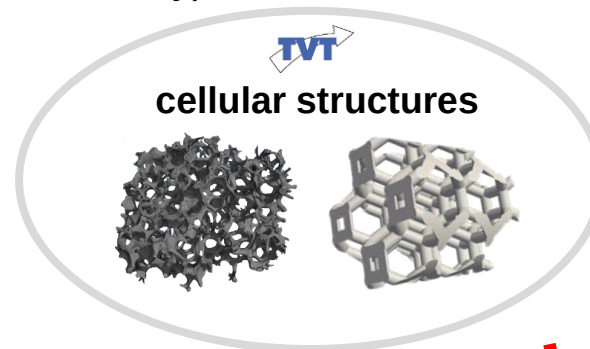
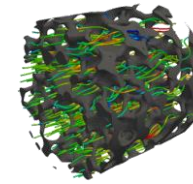
Experimental investigation

- single and multiphase flow
- pressure drop
- heat transfer coefficients
- heat conduction (effective thermal conductivity)
- flow distribution (via MRI)
- thermal radiation effects
- flow boiling of liquid CO₂



Numerical flow simulation

- single phase flow
- pressure drop
- heat transfer coefficients
- heat conduction (effective thermal conductivity)
- entrance effects
- flow regimes



Main Funding



process intensification using structured internals → enhancement of heat and mass transfer

- honeycomb structures



- packed beds



process intensification using structured internals → enhancement of heat and mass transfer

■ honeycomb structures

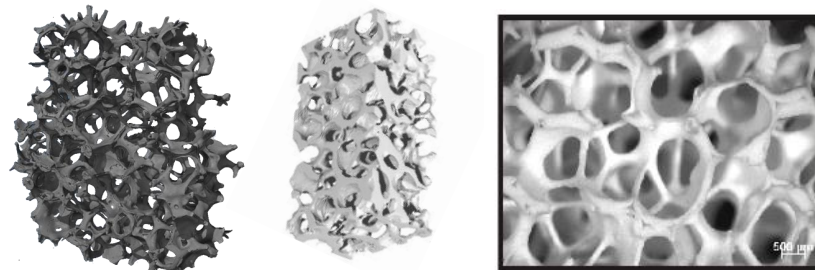


■ packed beds

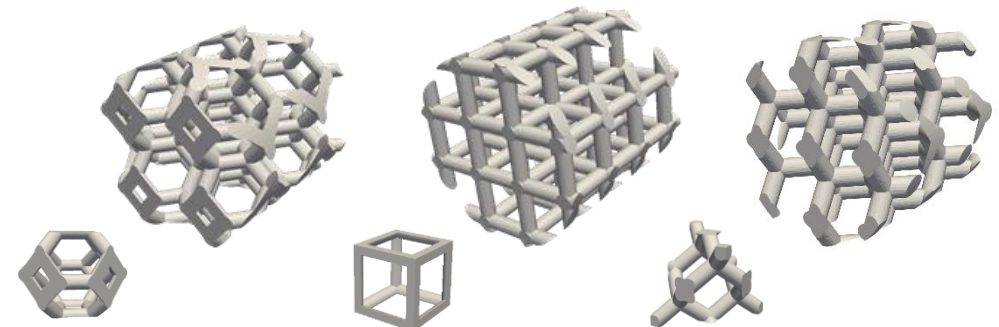


cellular structures

■ open-celled foams/ sponges



■ Periodic Open Cellular Structures (POCS)

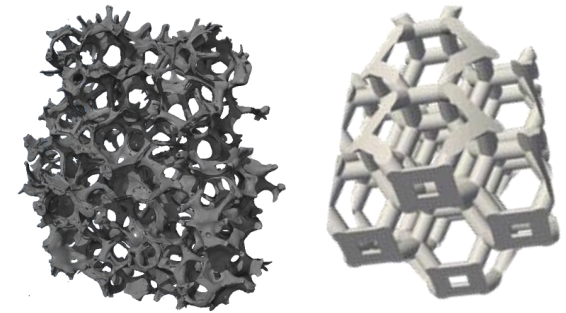


properties of cellular structures

- high porosities
- large specific surface area
- continuous solid (and fluid) phase
- POCS: great design freedom



- acceptable pressure drop
- high heat transfer coefficients
- good heat conduction

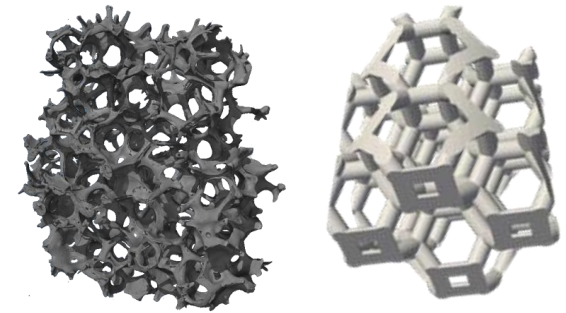


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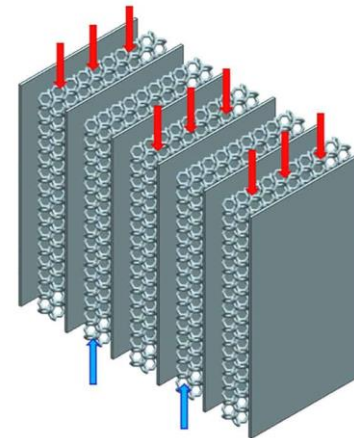


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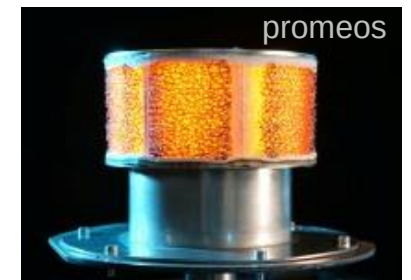


possible application fields

- chemical reactors
- light weight heat exchanger
- thermal and acoustic insulation
- solar tower plant
- porous burner
- shock absorber



Ceramics.org



designing process engineering systems

momentum transfer

pressure drop

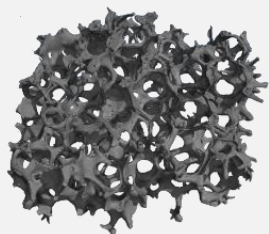
$$\Delta p / \Delta L$$



$$\frac{\Delta p}{\Delta L} = f \left(\begin{array}{l} \text{porosity,} \\ \text{unit cell,} \\ \text{cell design,} \\ \dots \end{array} \right) ?$$
$$h$$

heat transfer

heat transfer
coefficient h



analogy between momentum and heat transfer
(Generalized L  v  que Equation ¹)

applicable for sponges and POCS?



1: H. Martin, Chem. Eng. Process. 35 (1996) 301-310

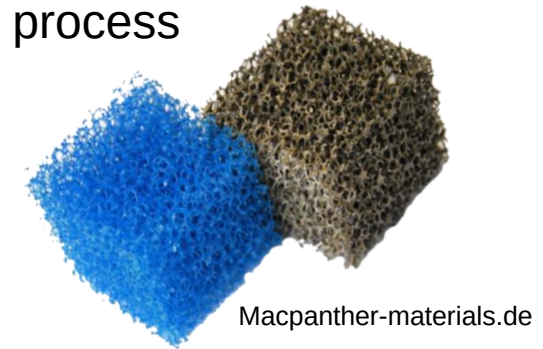
Sponge facts

characteristics

- typical cell densities: 5 ... 45 ppi (pores per inch)
- typical porosities: 75 ... 98 %
- stochastically random cell with broad size distribution
- many closed windows (= connecting faces between the cells)
- mass product, therefore comparatively cheap

manufacturing of sponges

- replica process
- casting process



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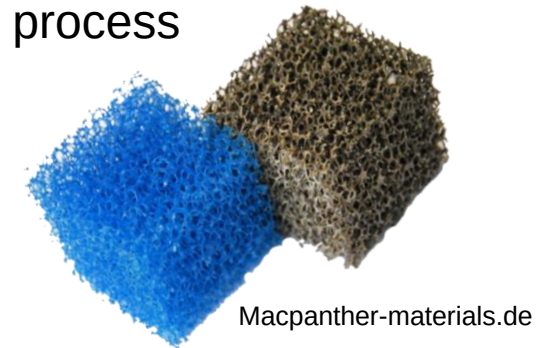
initial situation

- correlations only available for certain sponge types, but no cross-type systematics
- findings for one type are not easily transferable to other types

 **one correlation for each transport property applicable to all unit cell types is necessary**

manufacturing of sponges

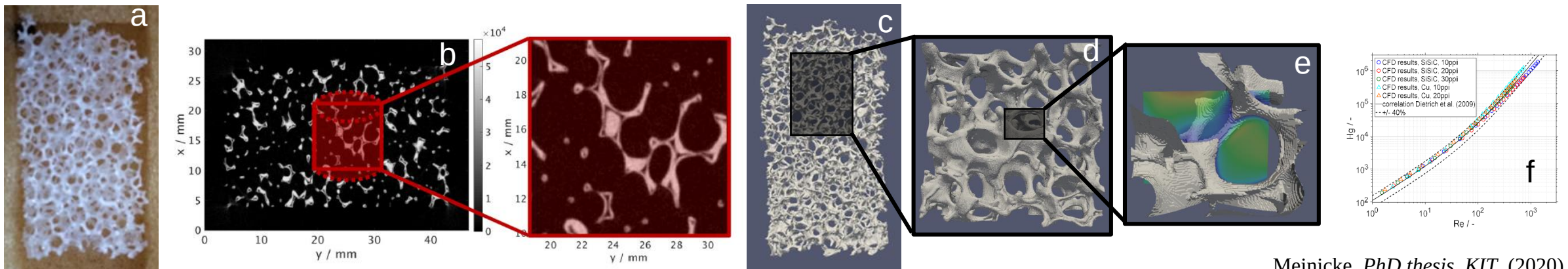
- replica process
- casting process



Sponges – numerical procedure

numerical simulation using real sponge geometries

- original sponge sample (a)
- μ CT measurement (b)
- geometrical reconstruction from μ CT data (c)
- CFD modelling geometry and definition of REV (d)
- CFD data evaluation (e)
- validation by experimental data on identical sponge sample (f)



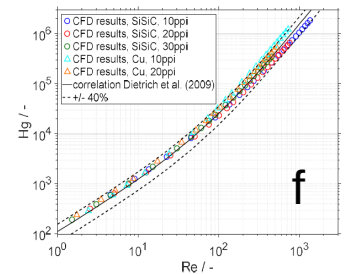
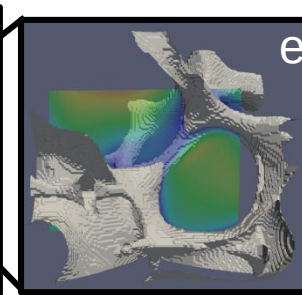
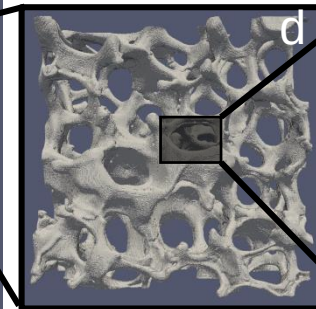
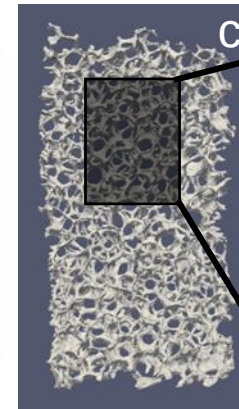
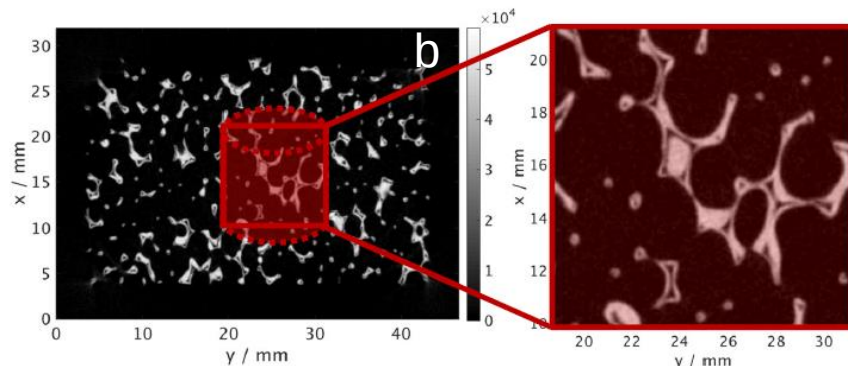
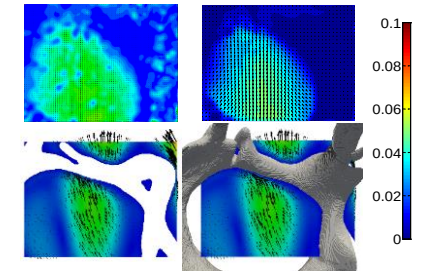
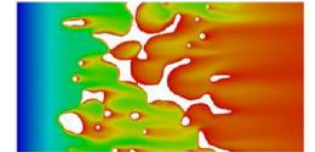
Meinicke, PhD thesis, KIT, (2020)

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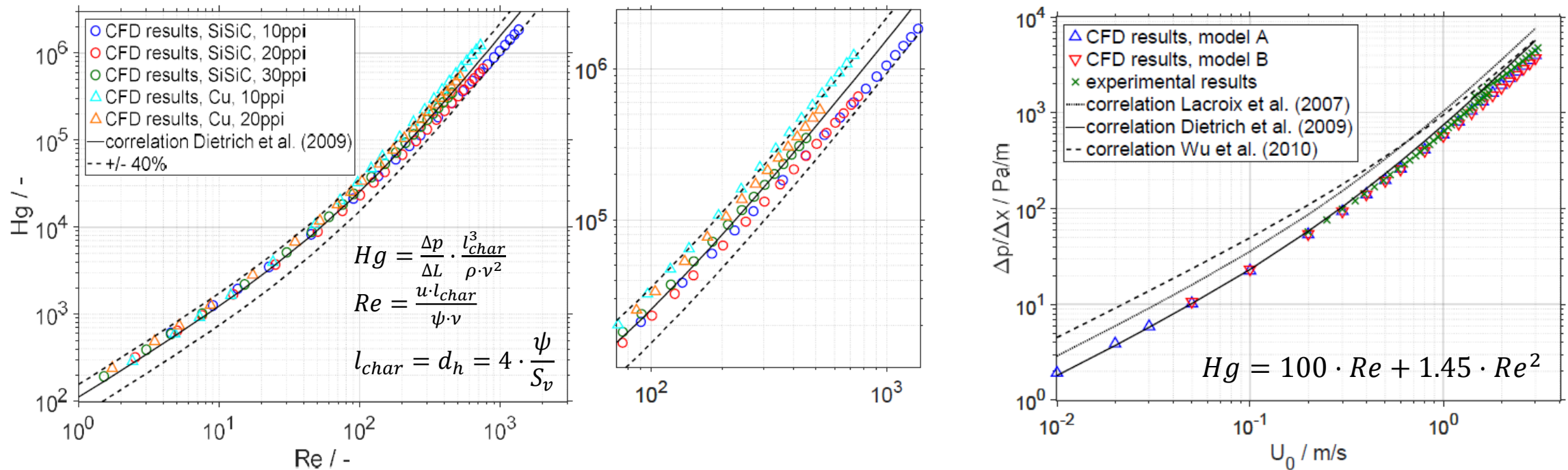
geometrical properties
heat transfer
pressure drop
flow regime
characterisation



Meinicke, *PhD thesis, KIT*, (2020)

Sponges – pressure drop

comparison of numerical results to experimental data

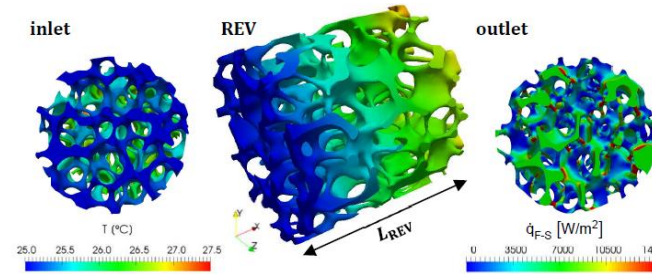
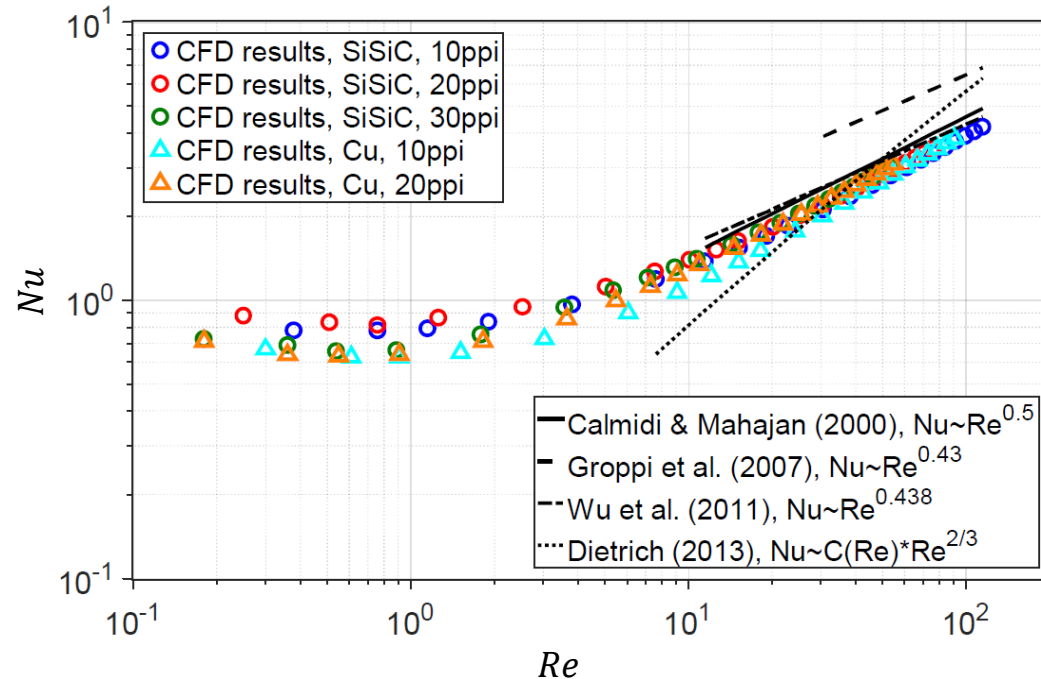


➔ CFD results fit well to the own correlation achieved from experimental data

Meinicke, PhD thesis, KIT, (2020)

Sponges – heat transfer

heat transfer correlation based on CFD data



- low Re: nearly constant
- higher Re: $Nu \sim Re^n$
- CFD data correspond to experimental data

Nusselt number

$$Nu = \frac{h \cdot d}{k_F}$$

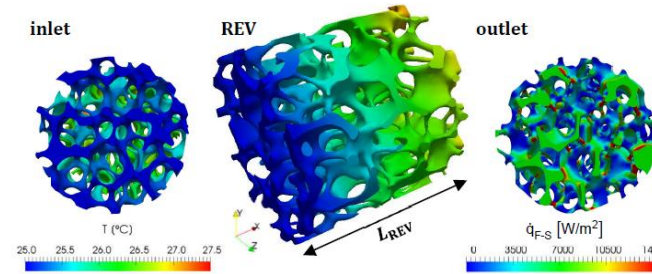
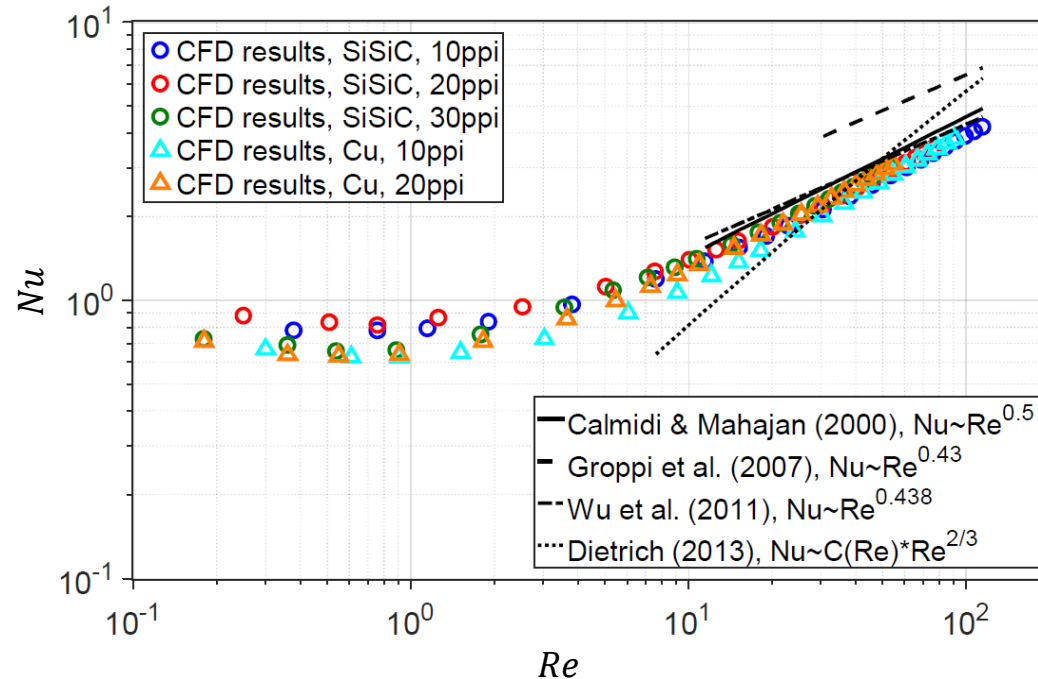
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d strut diameter

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Reynolds number

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d strut diameter

➡ data points for all sponge types coincide and can therefore be correlated

➡
$$Nu = \sqrt[3]{Nu_{min,sp}^3 + (C_{lam} \cdot Re^{0.5})^3} \quad \text{with} \quad Nu_{min,sp} = 0.71 \text{ and } C_{lam} = 0.39$$

Meinicke, PhD thesis, KIT, (2020)

Sponges – Generalized L  v  que Equation (GLE)

GLE = solution of velocity and temperature boundary layer equation for hydrodynamically developing flows by a similarity approach

$$h = C_{Le} \cdot \sqrt[3]{\frac{\tau \cdot Pr \cdot \rho_F \cdot k_F^3}{\mu_F^2 \cdot L_c}}$$

Nusselt number

$$Nu = \frac{h \cdot L_C}{k_F}$$

Hagen number

$$Hg = \frac{\Delta p \cdot L_C^3}{L \cdot \rho_F \cdot v_F^2}$$

characteristic
length

$$L_C = \frac{\pi \cdot d}{2}$$

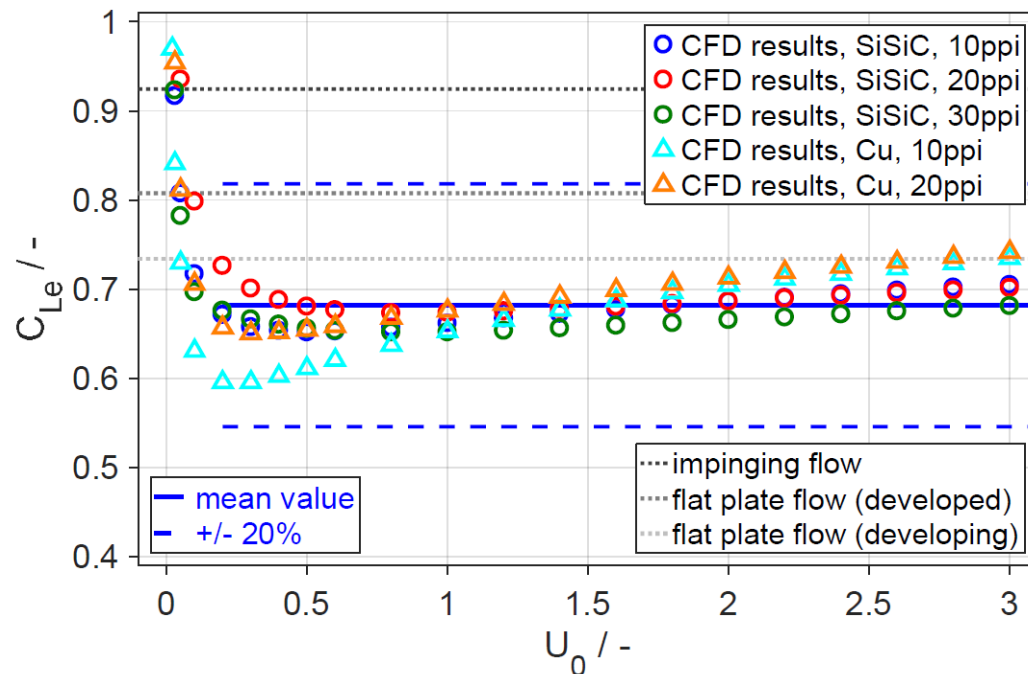
τ = local shear stress

Pr = Prandtl number

Meinicke, *PhD thesis, KIT*, (2020)

Sponges – Generalized L  v  que Equation (GLE)

GLE = solution of velocity and temperature boundary layer equation for hydrodynamically developing flows by a similarity approach



$$h = C_{Le} \cdot \sqrt[3]{\frac{\tau \cdot Pr \cdot \rho_F \cdot k_F^3}{\mu_F^2 \cdot L_C}}$$

- verified for different sponge types
(variation of solid material, cell density)
- GLE applicable if $C_{Le} = const.$

Nusselt number

$$Nu = \frac{h \cdot L_C}{k_F}$$

Hagen number

$$Hg = \frac{\Delta p \cdot L_C^3}{L \cdot \rho_F \cdot v_F^2}$$

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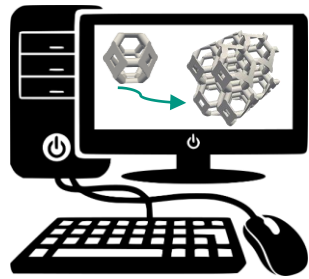
➔ GLE valid for „higher“ superficial velocities, where axial heat conduction is neglectable

➔ **heat transfer coefficients of sponges estimable from corresponding $\Delta p / \Delta L$**

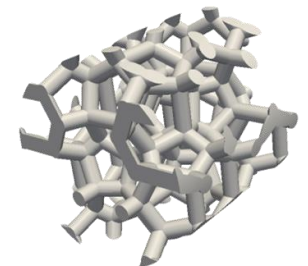
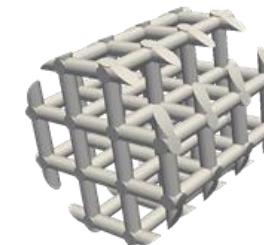
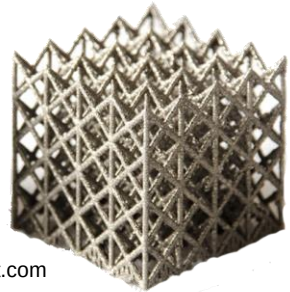
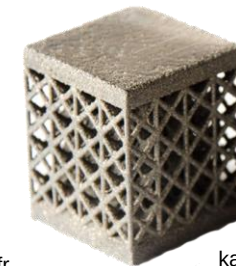
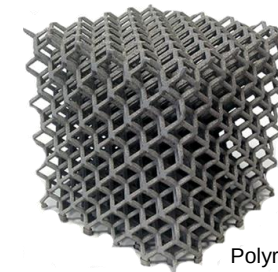
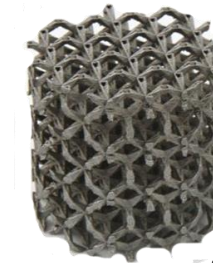
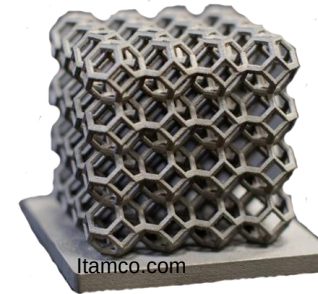
Meinicke, PhD thesis, KIT, (2020)

POCS – why particularly interesting?

Manufacturing of POCS

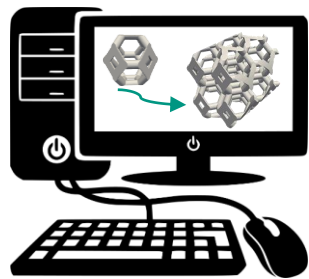


Additive
manufacturing

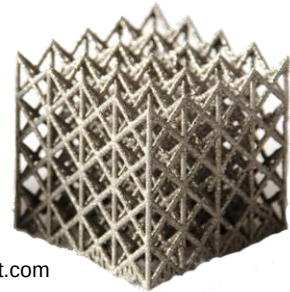
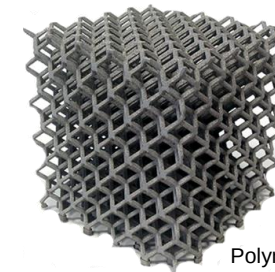
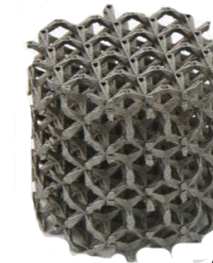
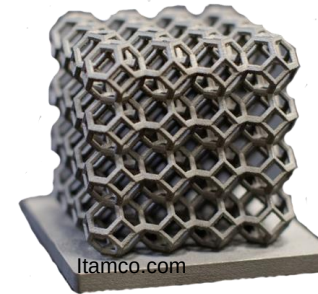


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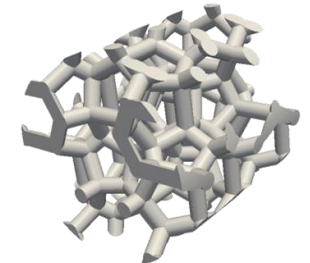
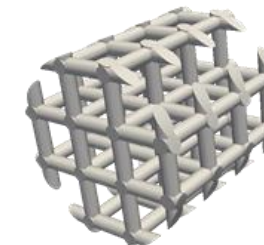


Additive
manufacturing



Advantages and Opportunities

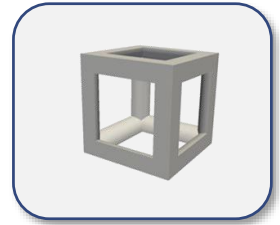
- large design freedom (unit cell, porosity, strut shape, ...)
 - variety of solid materials possible
 - geometries can be adapted to the installation space
 - direct wall connection possible
- ➡ hydrodynamics and heat transport can be directly influenced



POCS – heat transfer and pressure drop

initial situation

- correlations only available for certain unit cell geometries, but no cross-cell systematics!
 - findings are not easily transferable
 - prediction of arbitrary structures not reliable
- ➔ **one correlation for each transport property applicable to all unit cell types is necessary**

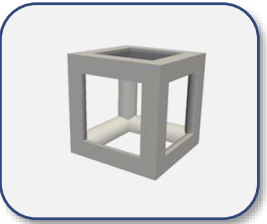


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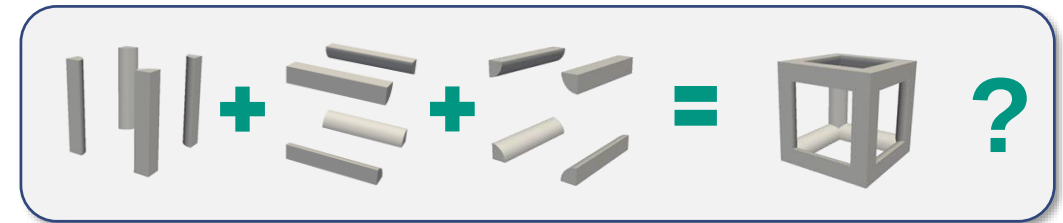
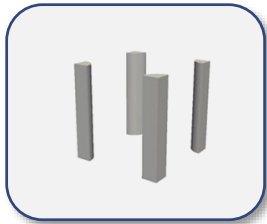
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➔ **one correlation for each transport property applicable to all unit cell types is necessary**



Hypothesis: Bottom – Up – Methodology

- unit cells are a geometrical superposition of strut arrangements
- correlations possible using a superposition of the contributions of respective strut arrangements
- Advantages:
 - strut arrangements = simple geometries
 - validated correlations are available
 - procedure applicable for any POCS



POCS – geometry specification

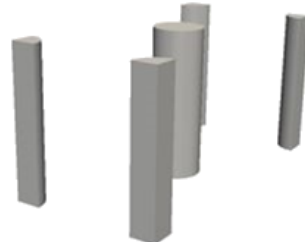
strut diameter
 $d = 0.64 \text{ mm}$

$s_T, s_L = \{2; 3; 4; 5\}$

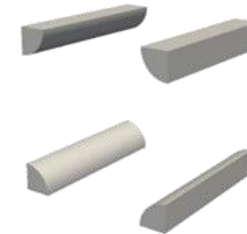
Struts



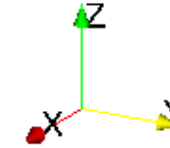
In-line struts



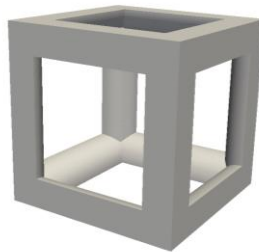
Staggered struts



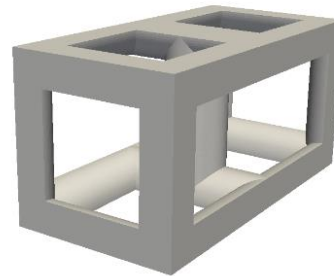
Inclined struts



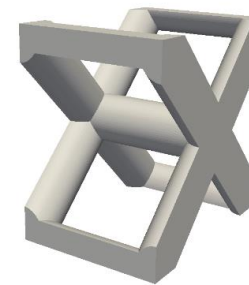
POCS



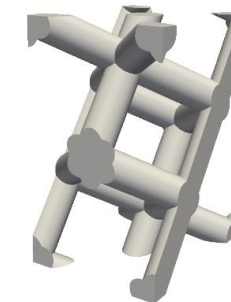
Cubic cell



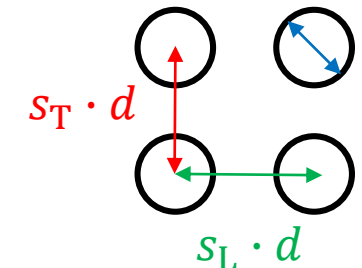
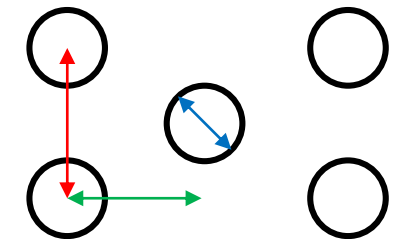
Staggered cubic cell



Inclined cubic cell

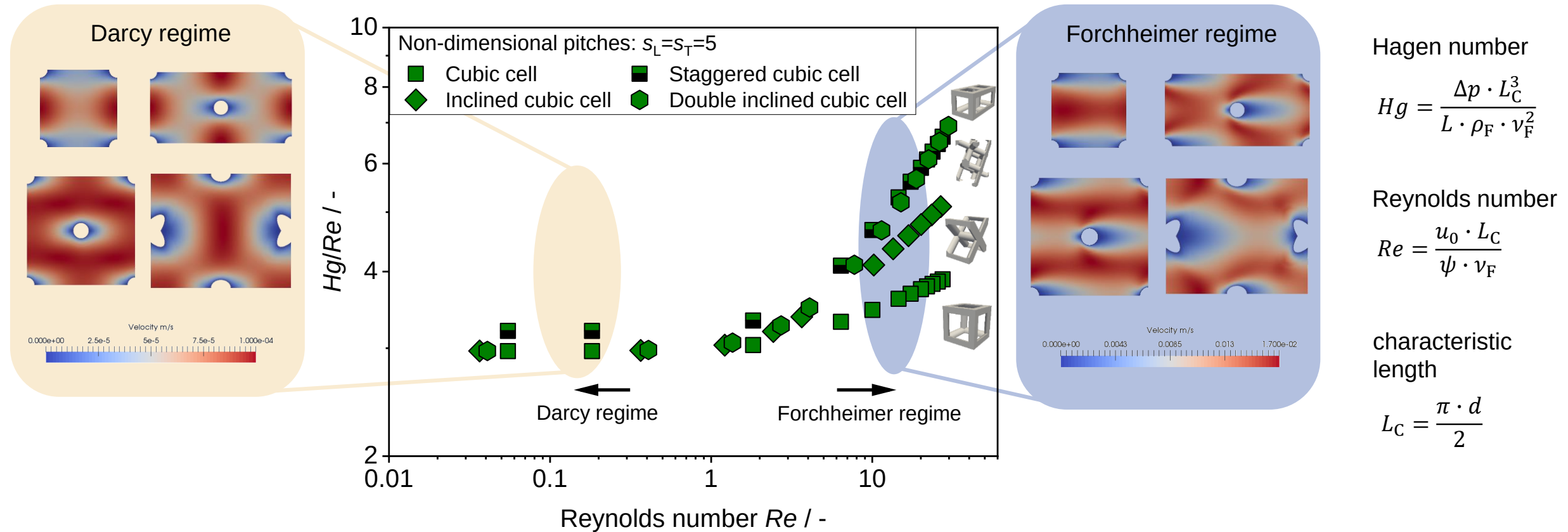


Double inclined cubic cell



Dubil et al., *Int. J. Heat Mass Transf.*, 200, 123546, (2023)

POCS – pressure drop

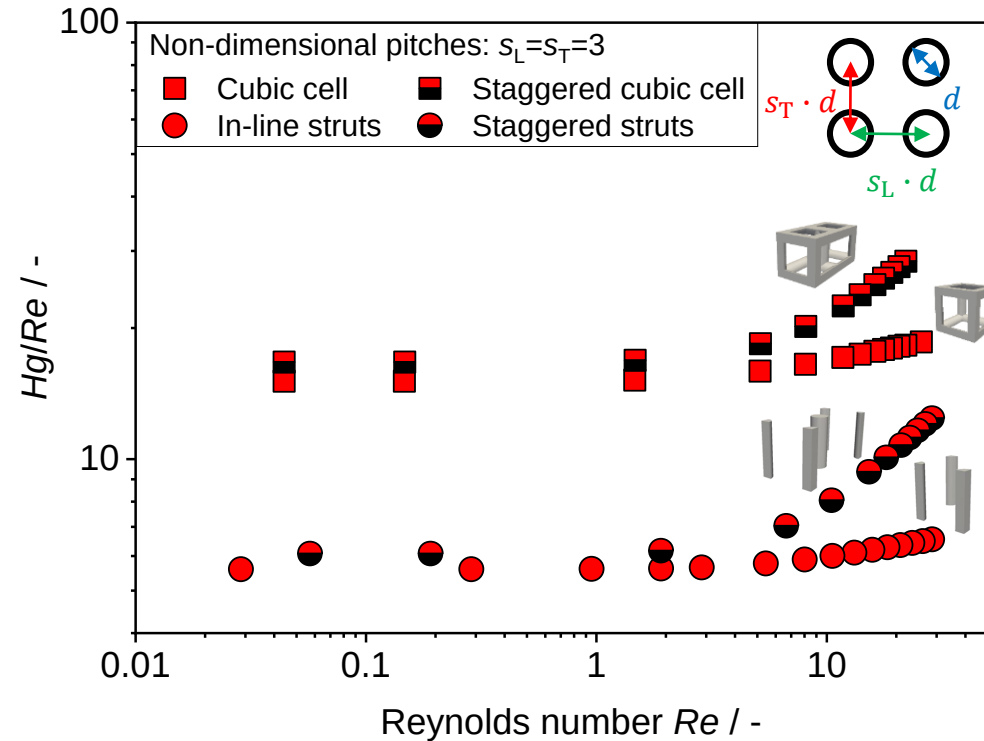
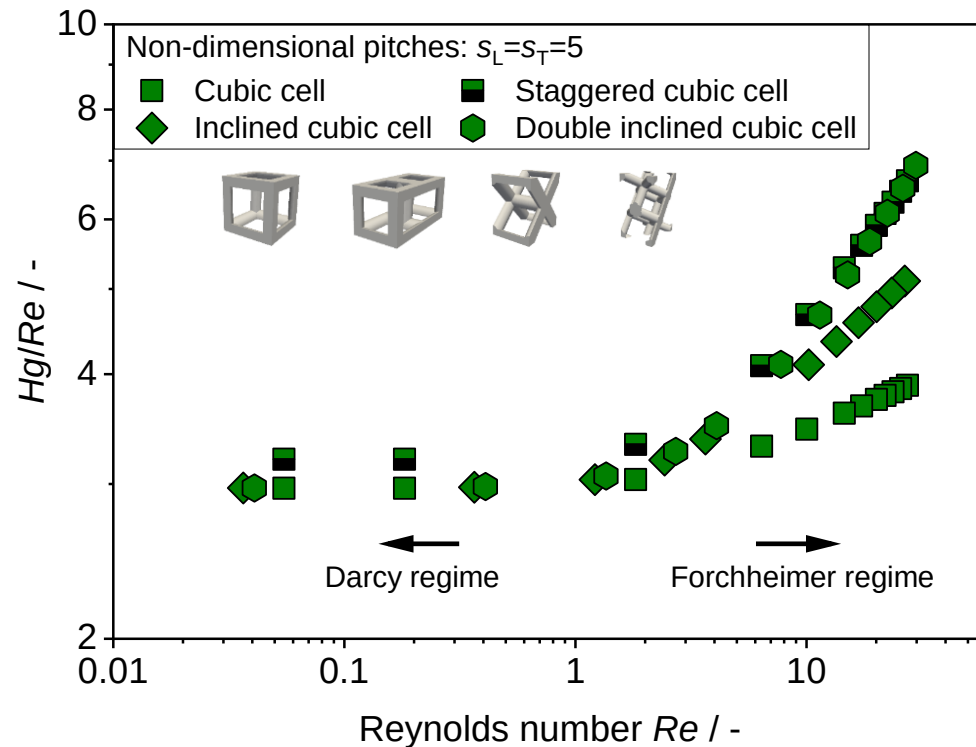


➡ linear (Darcy regime) and quadratic (Forchheimer regime) behavior

➡ significant influence of the unit cell geometry on pressure drop in the Forchheimer regime

Dubil et al.,
16th Int Conf Heat Transf
Fluid Mech Thermodyn, (2022)

POCS – pressure drop



Hagen number

$$Hg = \frac{\Delta p \cdot L_C^3}{L \cdot \rho_F \cdot v_F^2}$$

Reynolds number

$$Re = \frac{u_0 \cdot L_C}{\psi \cdot v_F}$$

characteristic length

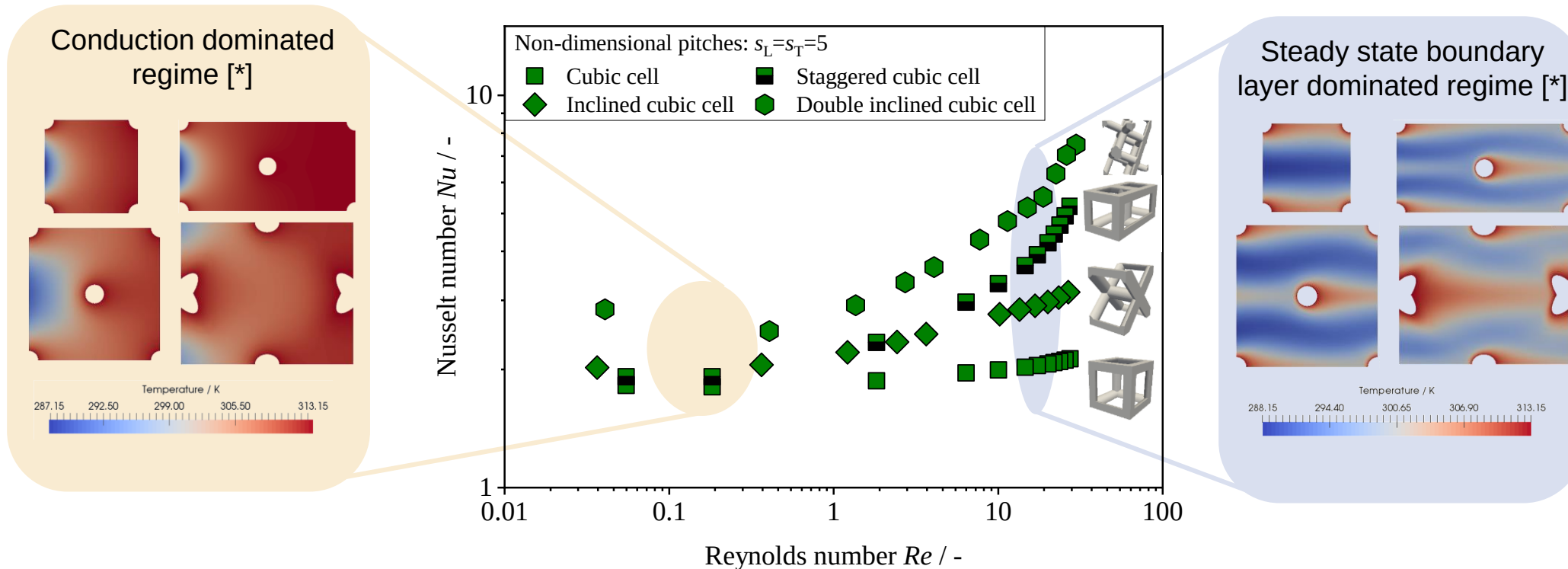
$$L_C = \frac{\pi \cdot d}{2}$$

➡ pressure drop increases when decreasing s_T due to decreasing porosity

➡ strut arrangements show same behaviour as corresponding POCS

Dubil et al., 16th Int Conf Heat Transf Fluid Mech Thermodyn, (2022)

POCS – heat transfer



Nusselt number

$$Nu = \frac{h \cdot L_C}{k_F}$$

Reynolds number

$$Re = \frac{u_0 \cdot L_C}{\psi \cdot \nu_F}$$

characteristic length

$$L_C = \frac{\pi \cdot d}{2}$$

- ➔ nearly constant behavior at low Re and $Nu \sim Re^n$ for higher Re
- ➔ orientation of the cubic cell has influence on heat transfer

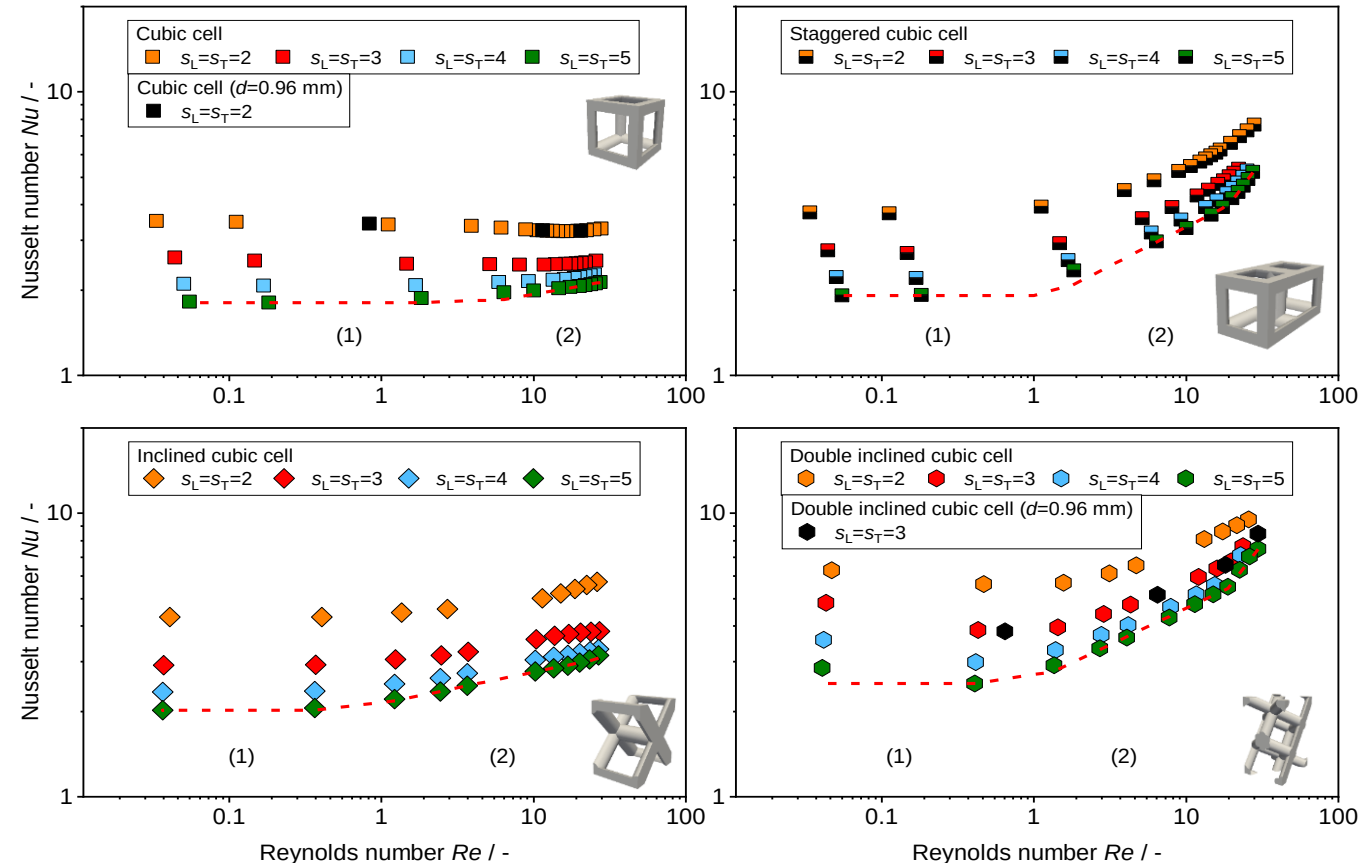
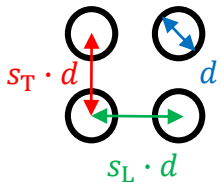
*Meinicke et al., *Int. J. Heat Mass Transf.*, 149, 119201 (2019)

Dubil et al., *16th Int Conf Heat Transf Fluid Mech Thermodyn.*, (2022)

POCS – heat transfer

Different Regimes [*]:

- Conduction dominated regime (1)
- Steady-state boundary layer dominated regime (2)



Nusselt number

$$Nu = \frac{h \cdot L_C}{k_F}$$

Reynolds number

$$Re = \frac{u_0 \cdot L_C}{\psi \cdot \nu_F}$$

characteristic length

$$L_C = \frac{\pi \cdot d}{2}$$

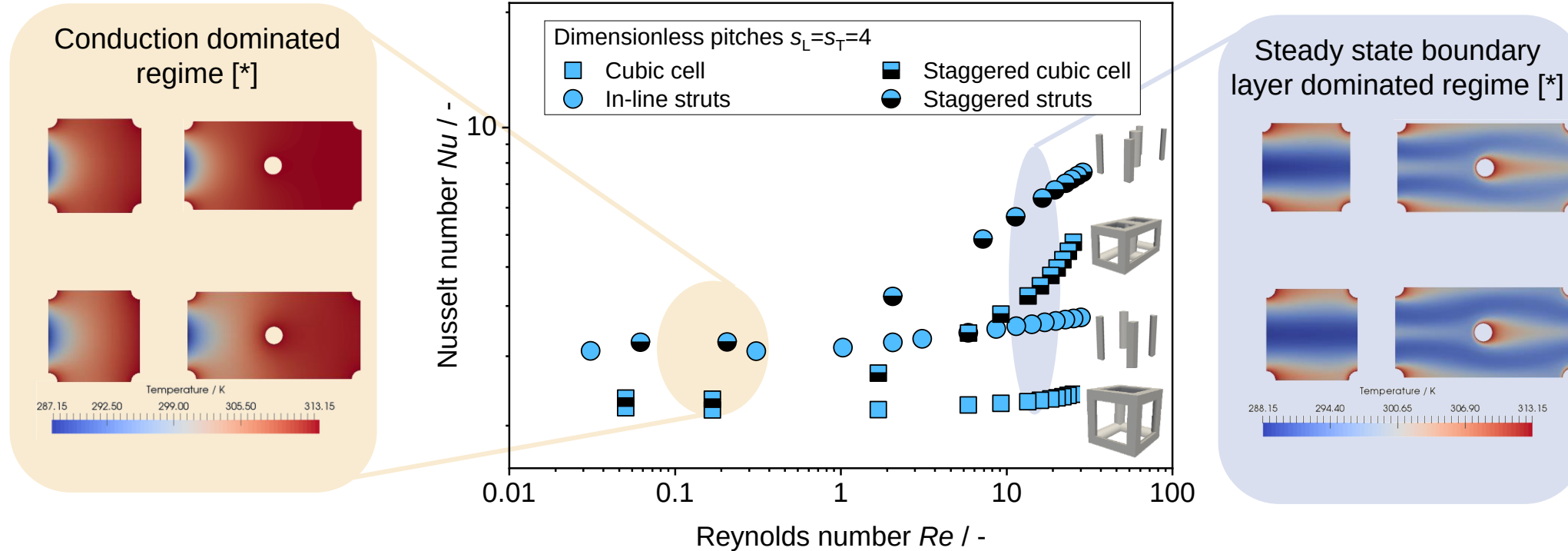
➡ distance (pitch) of the struts to each other influences heat transfer

➡ Can the observed differences between the unit cell geometries be quantified?

*Meinicke et al., *Int. J. Heat Mass Transf.*, 149, 119201 (2019)

Dubil et al., *Int. J. Heat Mass Transf.*, 200, 123546, (2023)

POCS – heat transfer



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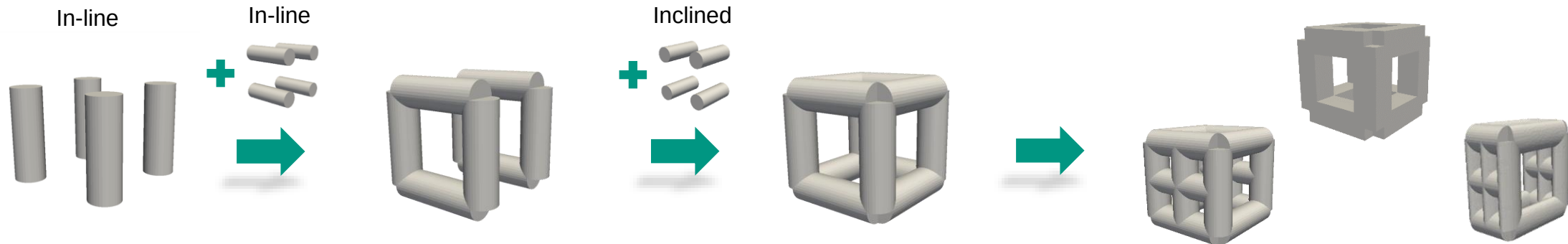
- ➔ strut arrangement influences heat transfer
- ➔ strut arrangements show same behavior as corresponding POCS

*Meinicke et al., *Int. J. Heat Mass Transf.*, 149, 119201 (2019)

Dubil et al., *Int. J. Heat Mass Transf.*, 200, 123546, (2023)

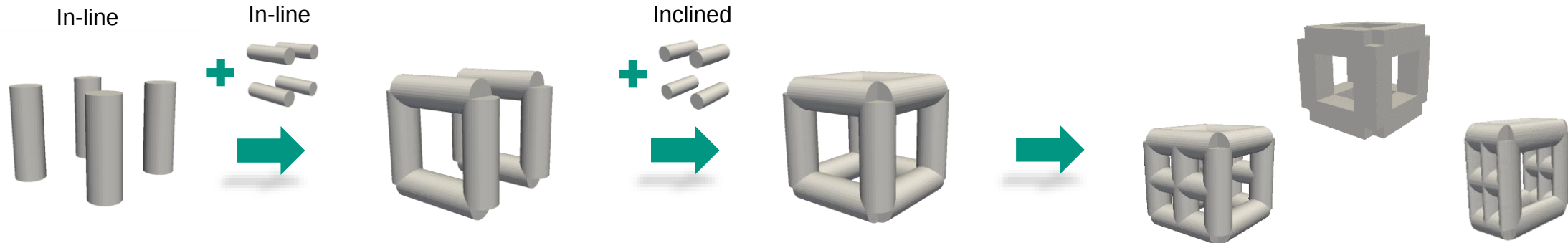
POCS – Bottom-Up Modeling Approach

Correlation of transport properties of any cells by a superposition of their structure-forming unit cells



POCS – Bottom-Up Modeling Approach

Correlation of transport properties of any cells by a superposition of their structure-forming unit cells



Mathematical procedure

■ Nusselt number
$$Nu_{\text{Cell}} = \sum_{i=1}^{N_{\text{Arr}}} Nu_i \cdot \frac{A_i}{A_{\text{Cell}}} \cdot f_{\text{Proj},i}$$

■ Hagen number
$$Hg_{\text{Cell}} = \sum_{i=1}^{N_{\text{Arr}}} Hg_i \cdot N_i$$

■ surface area of each strut arrangement $A_i = f_{\text{Node}} \cdot \pi \cdot d \cdot L \cdot N_{\text{Strut},i}$

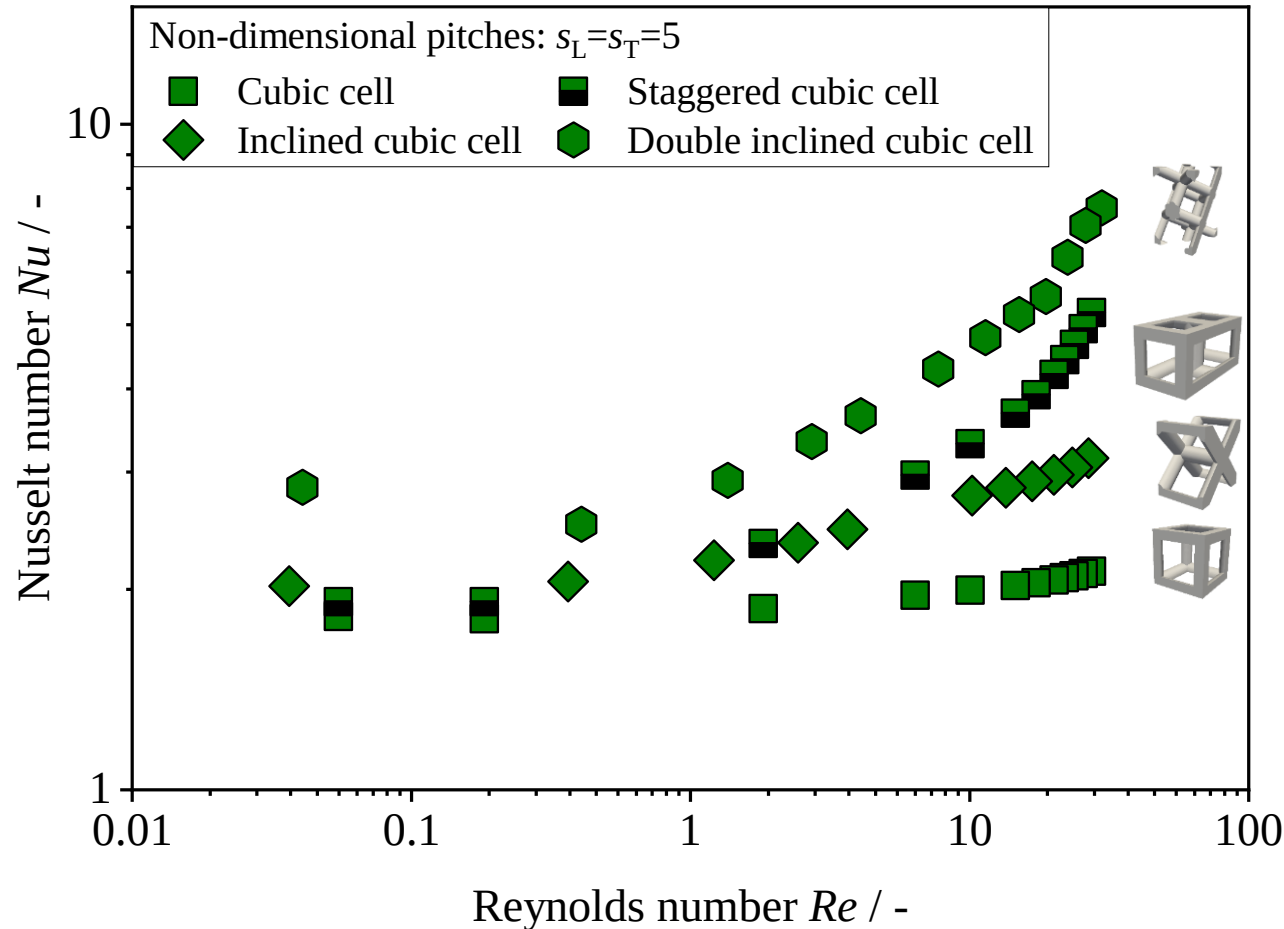
■ factors
$$f_{\text{Proj}} = \begin{cases} \frac{2}{\pi} & , \text{Inclined struts} \\ 1 & , \text{In - line and staggered struts} \end{cases}$$

$$f_{\text{Node}} = \begin{cases} 0.417, & s_T = s_L = 2 \\ 0.559, & s_T = s_L = 3 \\ 0.632, & s_T = s_L = 4 \\ 0.678, & s_T = s_L = 5 \end{cases}$$

Dubil et al., *Int. J. Heat Mass Transf.*, 200, 123546, (2023)

Dubil et al., *16th Int Conf Heat Transf Fluid Mech Thermodyn*, (2022)

POCS – Bottom-Up Modeling Approach



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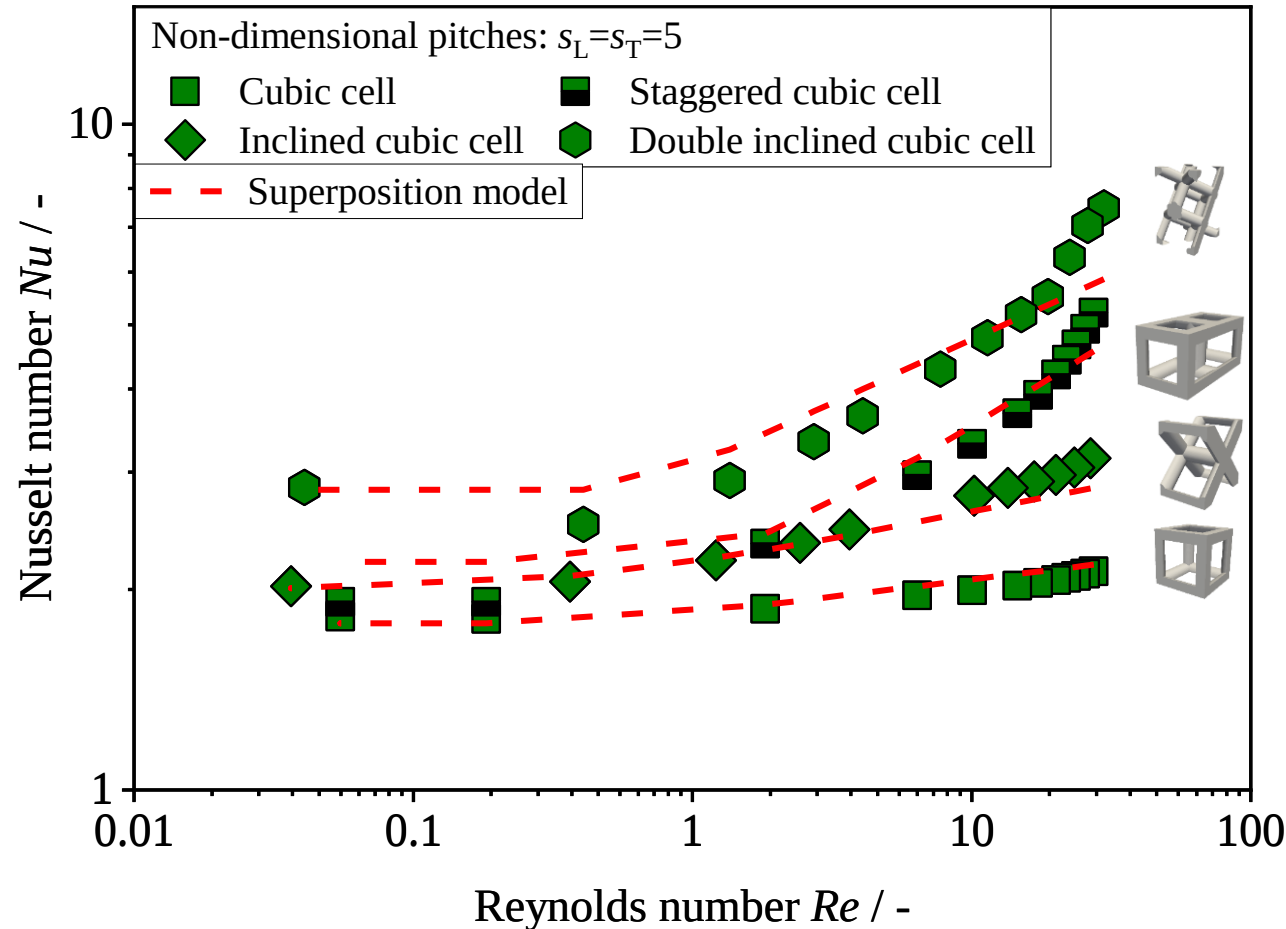
characteristic length

$$L_C = \frac{\pi \cdot d}{2}$$

Dubil et al., *Int. J. Heat Mass Transf.*, 200, 123546, (2023)

Dubil et al., *16th Int Conf Heat Transf Fluid Mech Thermodyn.*, (2022)

POCS – Bottom-Up Modeling Approach



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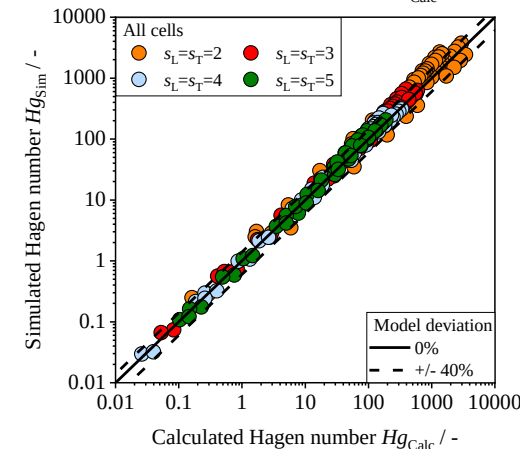
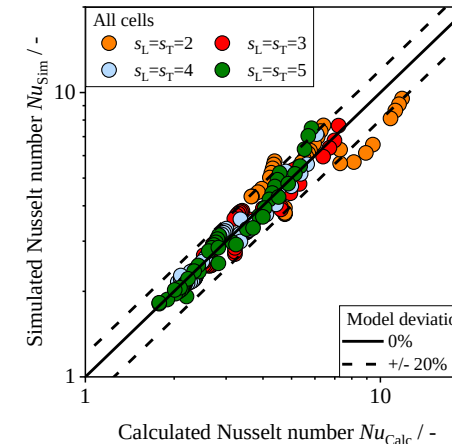
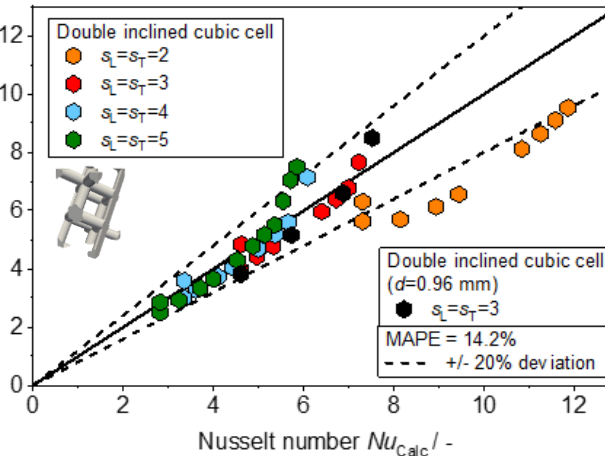
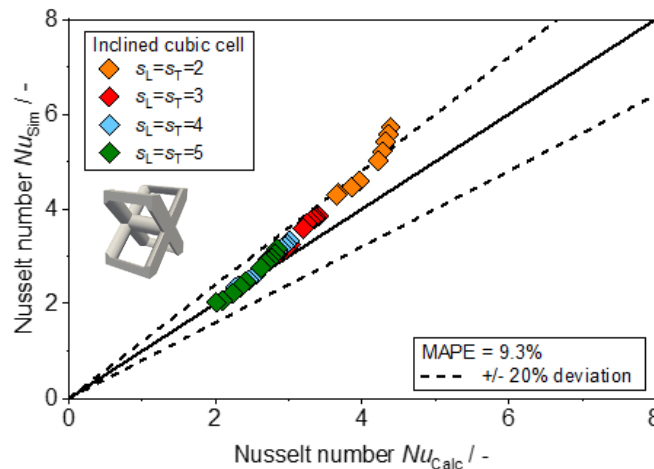
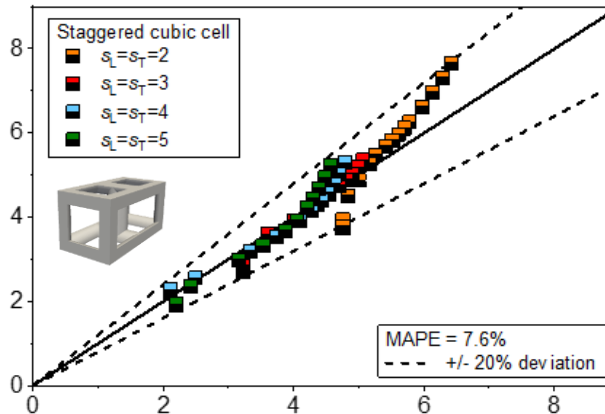
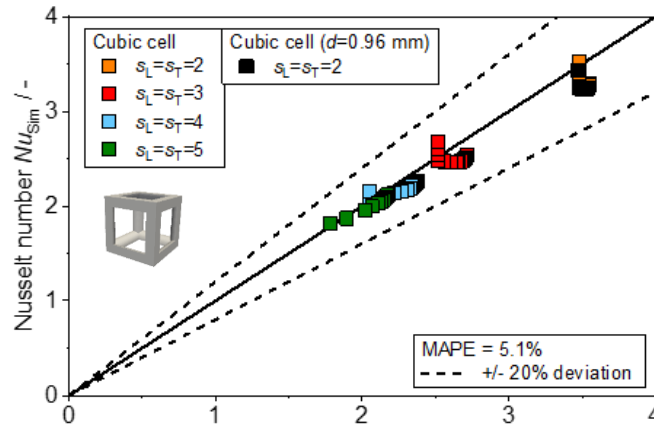
characteristic length

$$L_C = \frac{\pi \cdot d}{2}$$

➡ good agreement of simulation results with model for steady flow

Dubil et al., *Int. J. Heat Mass Transf.*, 200, 123546, (2023)
Dubil et al., *16th Int Conf Heat Transf Fluid Mech Thermodyn.*, (2022)

POCS – Bottom-Up Modeling Approach



$APE < 20\% = 92\%$

$MAPE = 15\%$

$APE < 40\% = 88\%$

$MAPE = 21\%$



model approach applicable for all investigated unit cells

Dubil et al., *Int. J. Heat Mass Transf.*, 200, 123546, (2023)

Dubil et al., *16th Int Conf Heat Transf Fluid Mech Thermodyn.*, (2022)

POCS – Generalized L  v  que Equation (GLE)

$$Nu = C_{Le} \cdot \sqrt[3]{x_{fric} \cdot Hg \cdot Pr \cdot \frac{2}{S_v \cdot \pi \cdot d}}$$

- Check for different unit cells with different $s_L = s_T$ - values
- GLE applicable if $C_{Le} = const.$

Nusselt number

$$Nu = \frac{h \cdot L_C}{k_F}$$

Hagen number

$$Hg = \frac{\Delta p \cdot L_C^3}{L \cdot \rho_F \cdot v_F^2}$$

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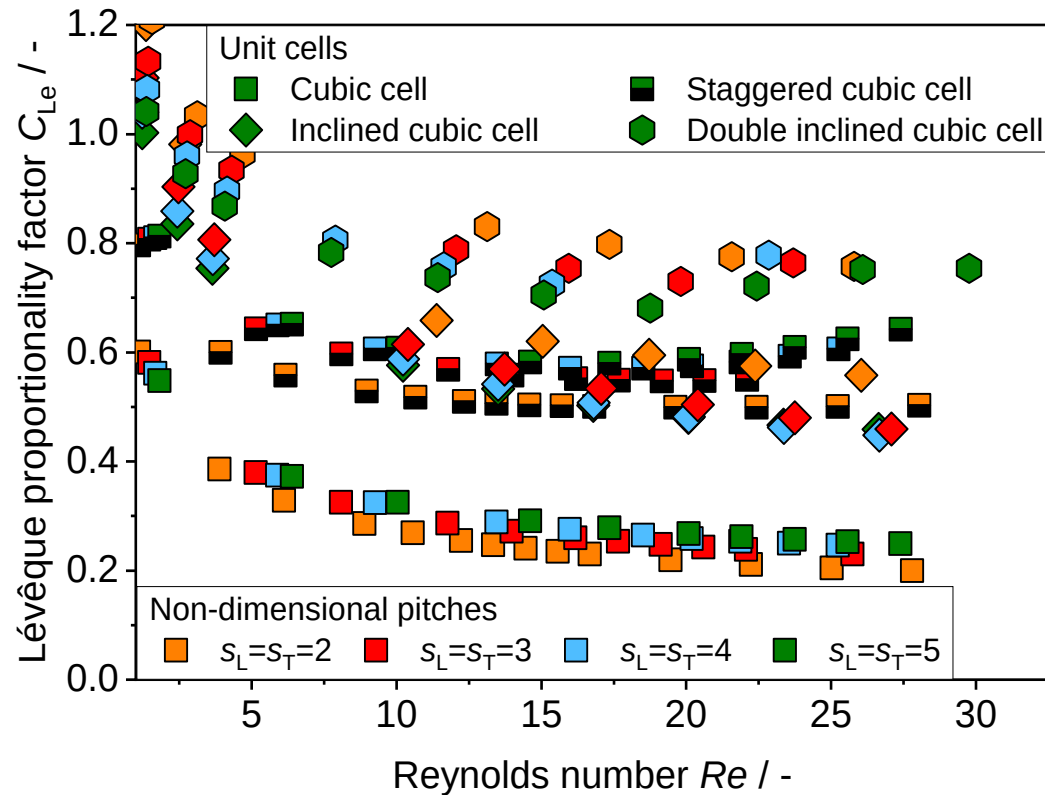
x_{fric} = friction factor

S_v = spec. surf. area

Pr = Prandtl number

Dubil et al., 16th Int Conf Heat
Transf Fluid Mech Thermodyn, (2022)

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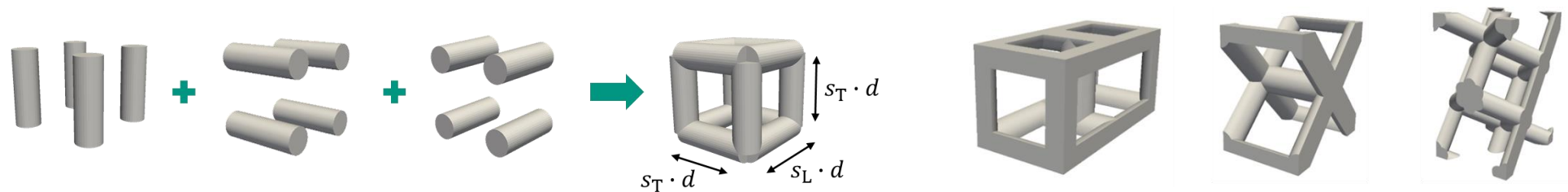
S_v = spec. surf. area

Pr = Prandtl number

- GLE valid for „higher“ Reynolds number, where axial heat conduction is neglectable
- heat transfer coefficients of POCS estimable from corresponding pressure drop

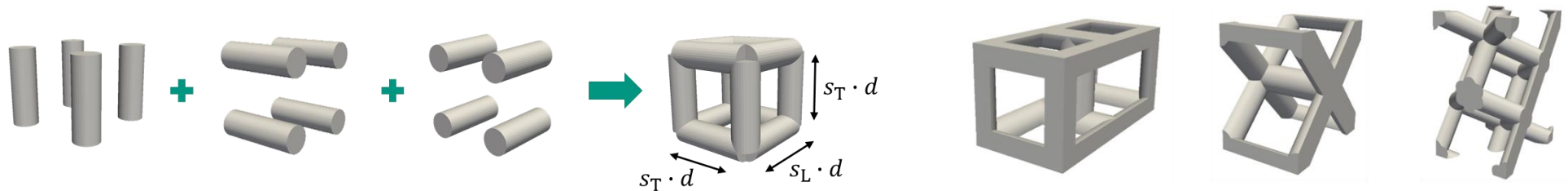
Dubil et al., 16th Int Conf Heat Transf Fluid Mech Thermodyn, (2022)

Hypothesis: transport parameters of complex cells (POCS) can be described as a superposition of the contributions of their respective strut arrangements



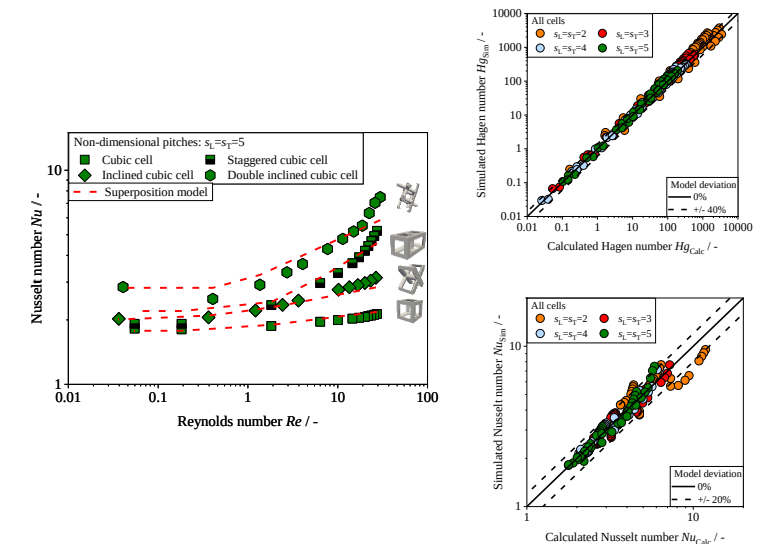
Summary

Hypothesis: transport parameters of complex cells (POCS) can be described as a superposition of the contributions of their respective strut arrangements



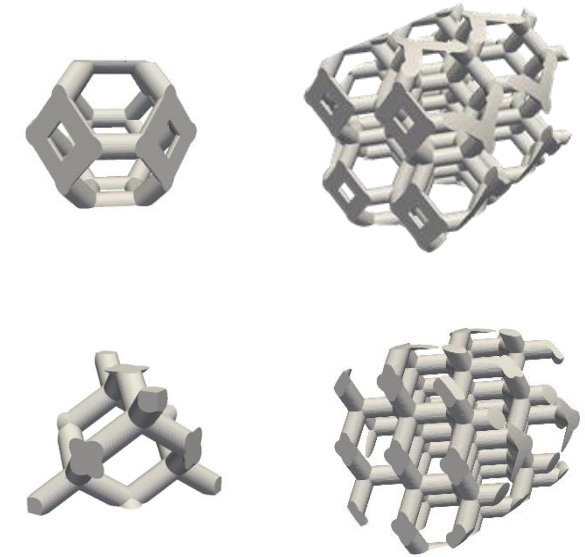
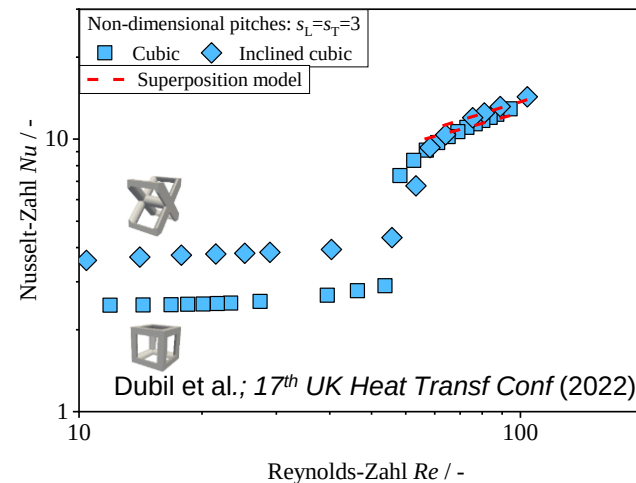
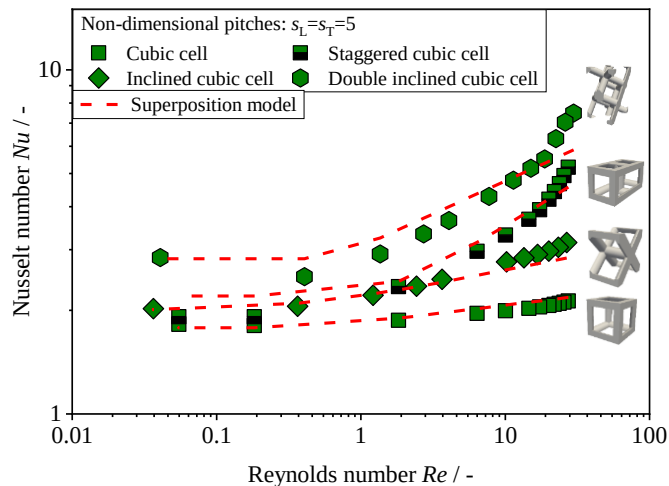
- strut arrangement significantly influences transport properties
- similar curve progressions for POCS and equivalent strut arrangements
- Hagen and Nusselt number as a function of Reynolds number of different cubic cells is well correlatable using one model approach

 **Hypothesis is verified for investigated cubic cells**

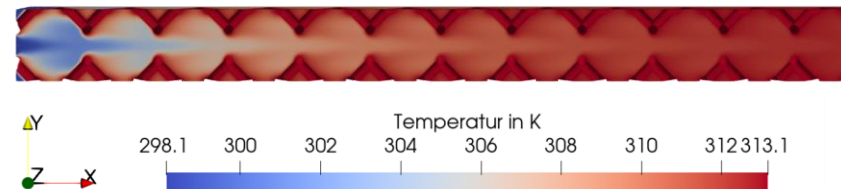


Outlook – current works

- Bottom-Up-Approach applicable for other types and complex structures?
- Bottom-Up-Approach applicable to unsteady flow?



- Thermal and hydrodynamic entrance length



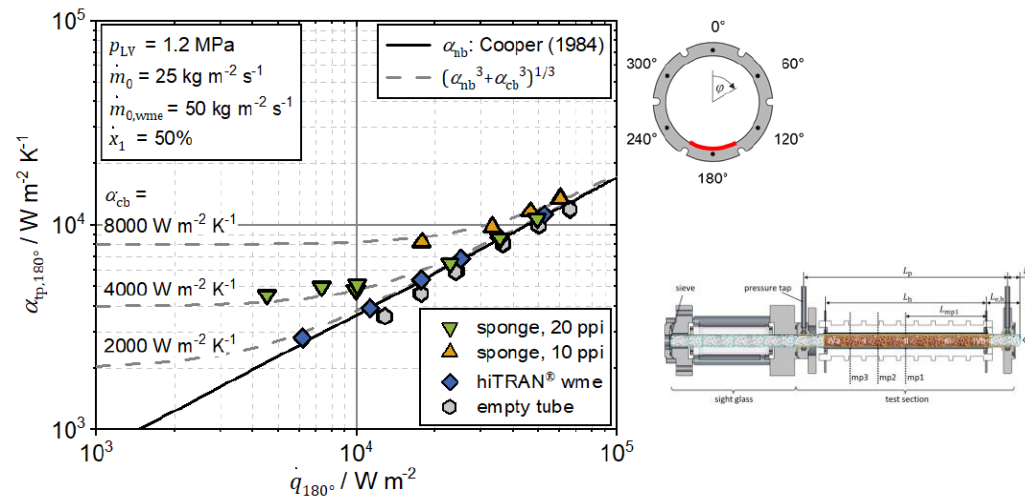
Knapp et al.; Dechema
Fachgruppentreffen WuS (2023)

Outlook – current works

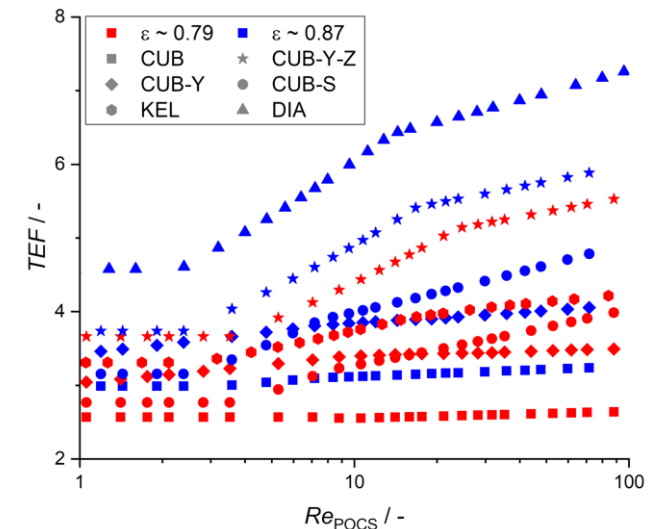
■ characterization of the structure performance

■ Theoretical: Thermal Enhancement Factor (TEF) → IHTC-17 | ID: 0064

- Experimental:
- Use as compact light weight heat exchanger for hybrid aircrafts
 - Use in flow boiling of liquid CO₂



Haertlé, PhD thesis, KIT, (2022)



Knapp et al., IHTC-17 | ID: 0064

Many thanks to ...

- ... you for your attention!
- ... my PhD-students Jonas Bender, Konrad Dubil, Sonja Haertlé, Katharina Knapp, Sebastian Meinicke for their excellent work!
- ... German Research Foundation (DFG),
Federal Ministry for Economic Affairs and Climate Action (BMWK),
Friedrich-und-Elisabeth-Boysen-Stiftung
for providing financial support of our research work!



DFG

Friedrich und Elisabeth
BOYSEN
Stiftung für Forschung und Innovation

Supported by:



Federal Ministry
for Economic Affairs
and Climate Action

on the basis of a decision
by the German Bundestag

